ATTACHMENT B
Amendments to the Specification

Please replace the paragraph at page 1, lines 3-11 with the following amended paragraph:

The present invention generally relates to a hydro mechanical clamping device which with one end thereof is intended to be mounted in a rotary or possibly a non-rotary machining device, such as a drilling machine, a milling machine, a lathe machine etc., and with the other end is intended to releasably hold a shaft tool, a work piece, a transition element, a hub or a similar object, such as a drill, a milling tool, a rotary saw blade, a grinding roll etc. ~~according to the preamble of claims 1 and 8~~.

Please replace the subheading at page 3, line 10 with the following amended subheading:

~~Aim~~ Summary of the ~~invention and most important features~~ Invention

Please replace the paragraph at page 3, lines 13-14 with the following amended paragraph:

This object is achieved by a hydromechanical clamping device as set forth in ~~claims 1 and 8~~ the claims.

Please replace the paragraph at page 8, lines 18-26 with the following amended paragraph:

~~I~~ In previously known devices the transmission of force is received via an inner sleeve. The construction of the present invention results in that received forces mostly is received through the outer sleeve. This is illustrated with arrows in fig. 2. Thanks to the

larger diameter of the outer sleeve it is capable of receiving considerably larger forces than an inner sleeve, which leads to that the tool 2 may work under a very high load without the arising of vibrations giving grooves in cut surfaces.

Please replace the paragraph at page 9, lines 10-25 with the following amended paragraph:

The sealing ring has a further function. As is shown in fig. 3, the sealing ring 20 is mounted closer to the pressure chamber 12 for mounting ~~that~~ than the pressure chamber 13 for dismantling. This has the effect that the friction between the piston 9 and the outer housing 7 is higher during mounting than during dismantling since a shorter part of the piston 9 can be lubricated by the hydraulic medium along the outer sleeve 7. A further effect that is obtained is that since the friction of the piston against the outer sleeve is lower during dismantling than during mounting, the pressure needed during dismantling is lower than the corresponding pressure that has been used during mounting. There is thus no risk that the necessary dismantling pressure is higher than an available pressure, which otherwise might be the case when a dismantling pressure equal to or higher than the mounting pressure is needed.